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2222) 2 (2 aCa L b

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$\sigma = \frac{P}{A} = \frac{W}{A} = \frac{2L}{A} = \frac{2}{A} \pi$, where $2 = \frac{2}{A}$
 $(2 \pi L C = (3.4.4)$
 Substituting (3.4.4)
 into (3.4.2) we have,
 $2 \frac{aCaAellp}{A} = \pi$ We
 can find the optimum
 solution by $0 = \frac{\partial}{\partial a} A$,
 by some operations
 leads to $0 = \frac{\partial}{\partial a} \frac{2 \pi L C}{A} = -$
 $\frac{2 \pi L C}{A^2} = \frac{\partial}{\partial a} \frac{2 \pi L C}{A} = \frac{2 \pi L C}{A^2} \pi$,
 therefore we have $2 \pi L C$
 $C a = \text{for } 0, > b a$
 Substitute it back to
 (3.4.4), we have $a C b$
 $= \frac{2 \pi L C}{A} (3.4.5)$ That
 means the optimum
 cross-section for

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elliptical shapes is a circle.

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$2222) 2 (2 aCa L b$
 $- = - = \pi$, where $2) 2$
 $(2 \pi L C = (3.4.4)$

Substituting (3.4.4)
into (3.4.2) we have,
 $22 aCaAellp - = \pi We$

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can find the optimum solution by $0 = \partial \partial a A$, by some operations leads to $0 \ 2 \ 22 \ 22 = -$
 $- = \partial \partial aC \ aC \ a A \ \pi$, therefore

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Two approaches are employed to find the solution. (i) Assume that the bending stress reaches the allowable allowable first and find the corresponding bending maximum bending moment. C T

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 $- = - = \pi$, where $2) 2$
 $(2 \pi L C = (3.4.4)$
Substituting (3.4.4)
into (3.4.2) we have,
 $22 aCaAellp - = \pi$ We
can find the optimum

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solution by $0 = \partial \partial a A$,
by some operations
leads to $0 \ 2 \ 22 \ 22 = -$
 $- = \partial \partial aC aC a A \pi$,
therefore

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06155. 2 2 2 3 4 2 1 2
 $2 = + + - = r$, The
central coordinate of
the circle is () 0, 5. 3 2
2 2, 2 3 4, = |) \ | \ ()
 $- + = c c y x$ Therefore

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we have the maximum and minimum stresses, respectively, 56155.5 and 06155.2 . $5.3 \max = +$
 $= + = r \times c \sigma 43845.1$
 06155.2 $5.3 \min = -$
 $= - = r \times c \sigma$ These are the same as we obtained above.

MAE 166A HW 2 Solution - 1 Consider a state of stress ...

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